

A research on Stability of Trimaran designed for Da Nang region of Vietnam

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Abstract: The authors have been working on designing a passenger trimaran according the region of Da Nang City, Viet Nam and research on its stability is one of the needed steps, although we all know trimarans have superior stability compared to a mono-hull ship of the same size. In this paper, the authors used The Dargnies and the Krilov method to determine the stability of the designed trimaran. In the calculation process, the AutoCad software was applied to draw and to take the values of the ship's stations areas, waterlines areas and the sinking parts volumes.

Keywords: trimaran, passenger ship, stability

I. INTRODUCTION

Da Nang City of Vietnam is known as the most attractive seaside city in this country. In Da Nang, most of tourist enterprises use converted ships from small wooden fishing boats, leading to many dangerous accidents. These ships are very old and were designed to serve the fishing industry, so they are not suitable for tourism industry, they particularly have low transverse stability. Along with the increasing demand for waterway tourism in this area due to the attraction of big events as well as the increasing tourism position of the city, many ships are ready to carry passengers in excess of the permitted number because profits, leading to the threatened safety of passengers. A typical example is the accident of Thao Van 2 ship in June 2016. The authors have been working on designing a passenger trimaran according the region of Da Nang City and research on its stability is one of the needed steps, although we all know trimarans have superior stability compared to a mono-hull ship of the same size.

II. APPROACH AND METHODS

There is a variety of methods for calculating the statical stability moment of ships. The Dargnies and the Krilov methods are used among the so-called "numerical" methods based on the same fundamental principle. The generally used above-mentioned methods are developed for the usual shaped ship hull and so it is advisable to apply them only in these cases. The simplifications introduced at the development of these methods are based on the recognition that waterlines differ slightly at two neighbouring inclinations and so do their statical moments and moments of inertia. In case of catamarans these differences are more significant and so the usual stability calculation methods result in higher error of the statical stability moment. Moreover, the amount of work of calculation, drawing and planimetry is much higher than for a single-hull ship. The authors combined this method of calculation with the application of modern technical facilities (AutoCad 3D).

2.1 Input data of the designed trimaran

- Length overall: $L_{max}=62m$;
- Designed length: $L_{tk}=60m$;
- Breath overall: $B_{max}=19.8m$;
- Designed breath: $B_{tk}=18m$;
- Height: $D=4.5m$;
- Draught: $d=2m$;
- Volume full coefficient: $C_B=0.645$;
- Waterline full coefficient: $C_w=0.22$;
- Midship section full coefficient: $C_M=0.69$.

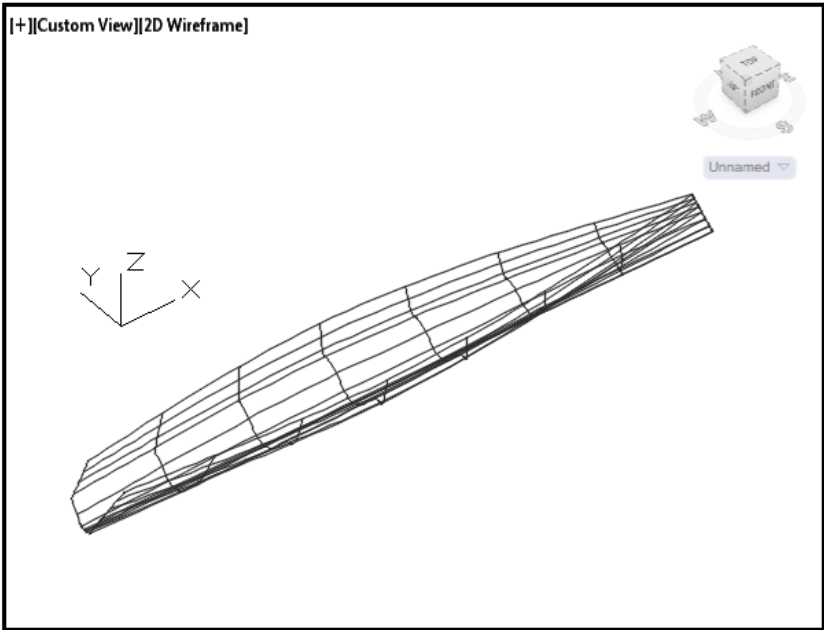


Figure 1 Middle hull shape

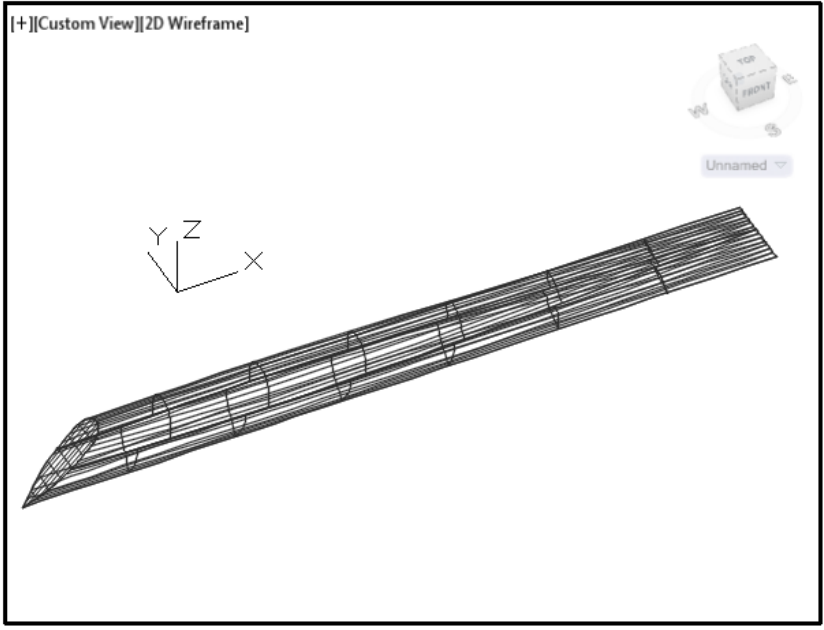


Figure 2 Side hull shape

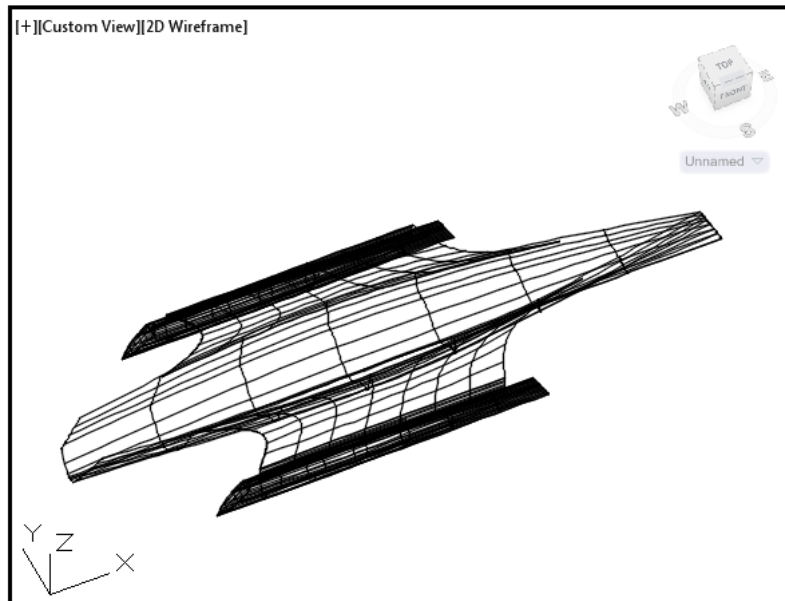


Figure 3 Designed trimaran hulls shape

2.2 Drawing Chebyshev station and constructing constant volume tilt waterlines

The authors return to do the followings:

- Draw the Chebyshev station of the trimaran fully side to side;

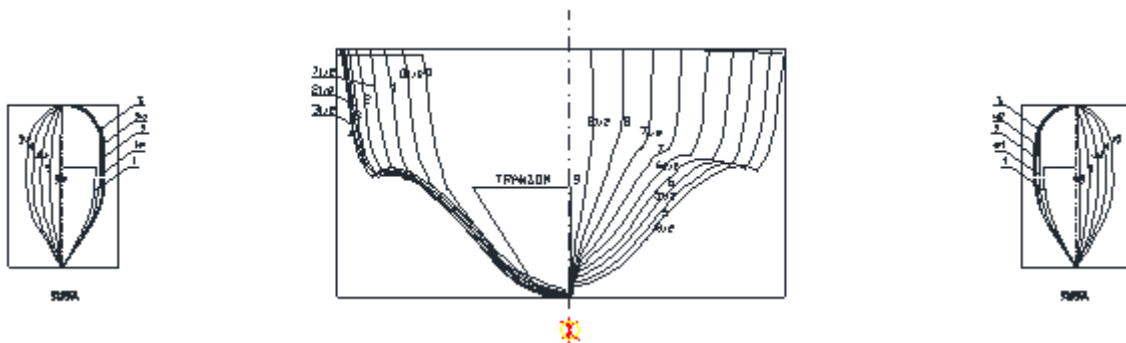


Figure 4 Chebyshev stations of the designed trimaran

- Trace original waterline (when $\theta = 0^\circ$) WL_0 ;
- Through center O of the original waterline, draw the first auxiliary waterline under the inclination angle θ (equal to 10°) WL_1
- Through the center of rotation (determined by the interpolation method), the original waterline is converted into a new waterline, where the displacement equal the displacement at the position of the ship balanced with WL_0 .
- Find the center of rotation to bring the WL_1 waterline back to a horizontal state with the rotation angle of -10° ;
- Find floating center coordinates
- Find the stability lever arm (YB)
- From there determine the next waterlines WL_1, WL_2, WL_3 , etc.
- Determine the center of rotation of WL_2, WL_3 by drawing the waterline WL_2' (take $\theta - 1^\circ$), the center of rotation is the intersection of WL_2 and WL_2' .
- Repeat the same for the next waterlines.

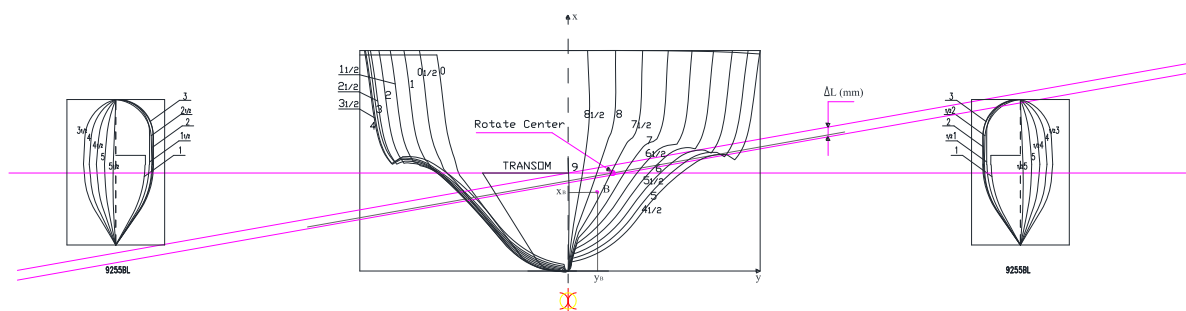


Figure 5 Determining the ship's stability lever arms

2.3 Determining characteristics of tilt waterlines at each tilt angle

The authors used AutoCad 3D software to determine the waterlines at the respective tilt angles:

- Initially, set the plane through the designed waterline, when the ship was in a balance position;
- Rotate the waterline with the corresponding tilt angle (from 10° to 90° , use Rotate3D);
- Alternately move the rotated waterline according to the direction perpendicular to the waterline for one segment L, find 4 locations in which there are 2 locations of measured displacement (using Massprop) is greater than the designed displacement and 2 locations with the measured displacement is less than the designed displacement.
- Graph from 4 obtained values, use the interpolation method to find the waterline at each tilt angle.

III. CALCULATION RESULTS

3.1 At the tilt angle of 10°

Table 1 Static characteristics when $\theta = 10^\circ$

| Tilt angle | 10 | |
|----------------|--------------------|--------------|
| ΔL (m) | ∇ (m^3) | Δ (T) |
| 0 | 326.50 | 334.66 |
| -50 | 314.80 | 322.67 |
| -100 | 303.20 | 310.78 |
| -150 | 291.80 | 299.10 |

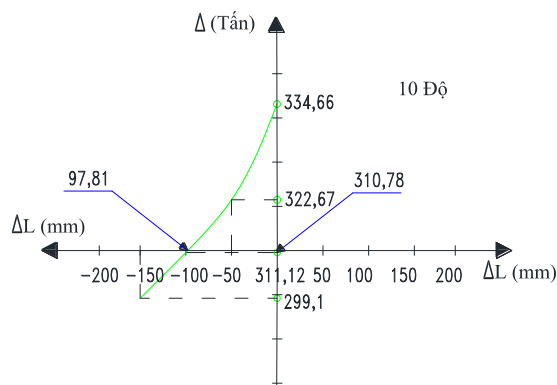


Figure 6 Interpolation graph when $\theta = 10^\circ$

3.3 At the tilt angle of 20°

Table 2 Static characteristics when $\theta = 20^\circ$

| Tilt angle | 20 | |
|----------------|--------------------|--------------|
| ΔL (m) | ∇ (m^3) | Δ (T) |
| 0 | 357.35 | 366.28 |
| -100 | 333.82 | 342.17 |
| -250 | 299.52 | 307.01 |
| -300 | 288.38 | 295.59 |

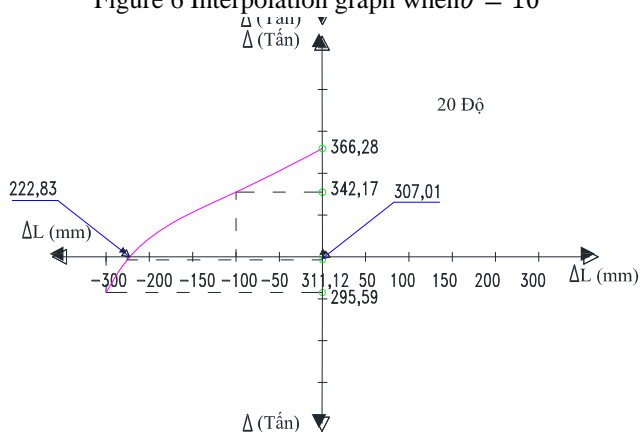


Figure 7 Interpolation graph when $\theta = 20^\circ$

3.3 At the tilt angle of 30°

Table 3 Static characteristics when $\theta = 30^\circ$

| Tilt angle | 30 | |
|----------------|----------------------------|--------------|
| ΔL (m) | ∇ (m ³) | Δ (T) |
| -200 | 355.10 | 363.98 |
| -400 | 310.63 | 318.40 |
| -500 | 289.07 | 296.30 |
| -600 | 268.01 | 274.71 |

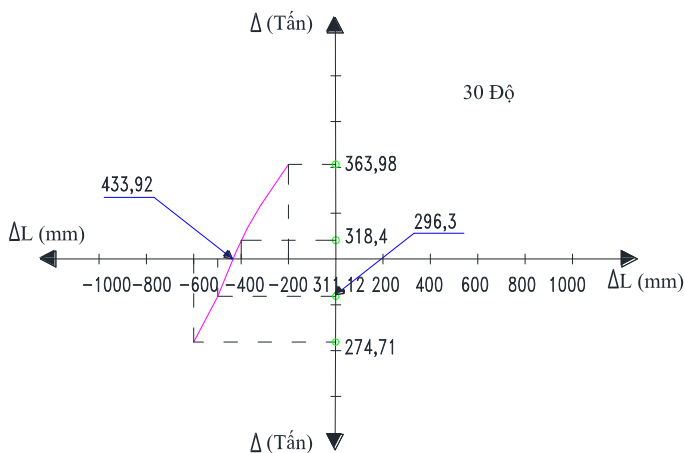


Figure 8 Interpolation graph when $\theta = 30^\circ$

3.4 At the tilt angle of 40°

Table 4 Static characteristics when $\theta = 40^\circ$

| Tilt angle | 40 | |
|----------------|----------------------------|--------------|
| ΔL (m) | ∇ (m ³) | Δ (T) |
| -400 | 326.53 | 334.69 |
| -500 | 306.89 | 314.56 |
| -600 | 287.29 | 294.47 |
| -700 | 192.27 | 197.08 |

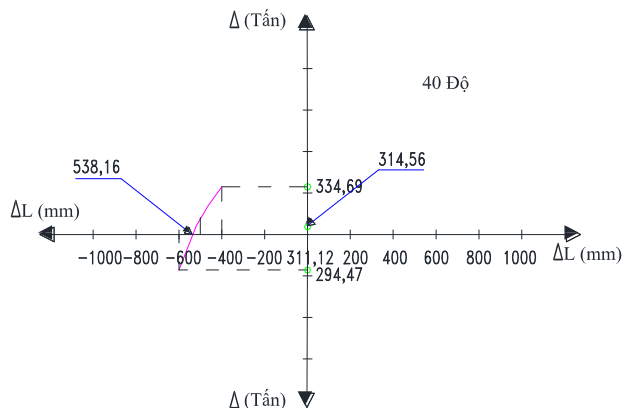


Figure 9 Interpolation graph when $\theta = 40^\circ$

3.5 At the tilt angle of 50°

Table 5 Static characteristics when $\theta = 50^\circ$

| Tilt angle | 50 | |
|----------------|----------------------------|--------------|
| ΔL (m) | ∇ (m ³) | Δ (T) |
| -400 | 375.74 | 385.13 |
| -500 | 360.23 | 369.24 |
| -600 | 344.47 | 353.08 |
| -800 | 313.06 | 320.89 |
| -1000 | 281.72 | 288.76 |
| -1100 | 266.15 | 272.80 |

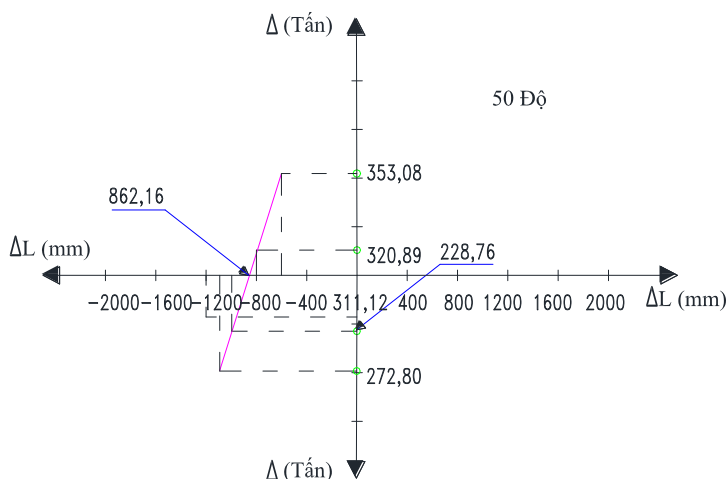


Figure 10 Interpolation graph when $\theta = 50^\circ$

3.6 At the tilt angle of 60°

Table 6 Static characteristics when $\theta = 60^\circ$

| Tilt angle | 60 | |
|------------|----------|----------|
| Δ_L | ∇ | Δ |
| -1200 | 321.40 | 329.44 |
| -1300 | 309.80 | 317.55 |
| -1400 | 298.31 | 305.77 |
| -1500 | 286.85 | 294.02 |

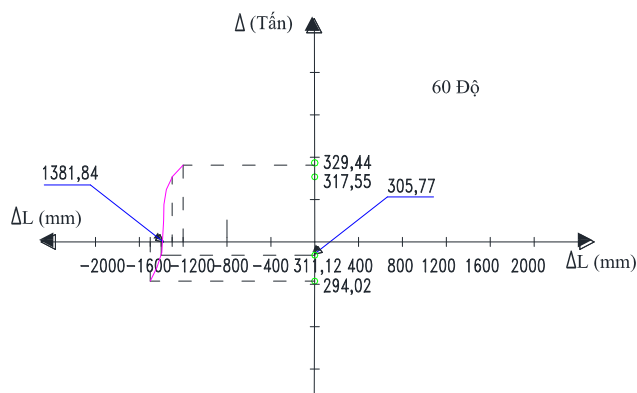


Figure 11 Interpolation graph when $\theta = 60^\circ$

3.7 At the tilt angle of 70°

Table 7 Static characteristics when $\theta = 70^\circ$

| Tilt angle | 70 | |
|------------|----------|----------|
| Δ_L | ∇ | Δ |
| -1400 | 368.70 | 377.92 |
| -1800 | 337.98 | 346.43 |
| -2400 | 294.43 | 301.79 |
| -2800 | 266.87 | 273.54 |

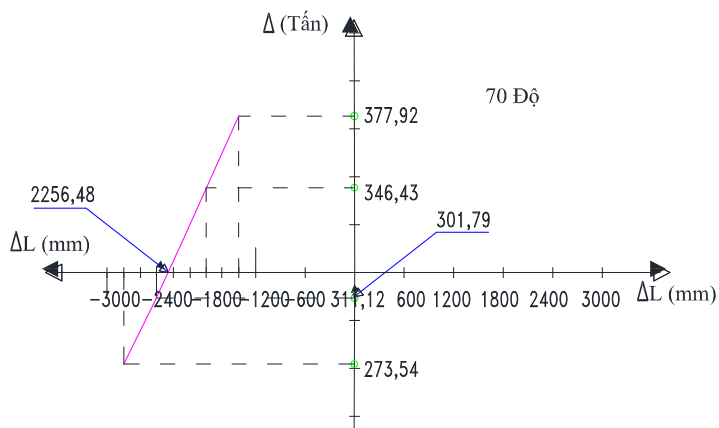


Figure 12 Interpolation graph when $\theta = 70^\circ$

3.8 At the tilt angle of 80°

Table 8 Static characteristics when $\theta = 80^\circ$

| Tilt angle | 80 | |
|------------|----------|----------|
| Δ_L | ∇ | Δ |
| -2000 | 402.39 | 412.45 |
| -3000 | 320.60 | 328.62 |
| -4000 | 234.12 | 239.97 |
| -5000 | 146.61 | 150.28 |

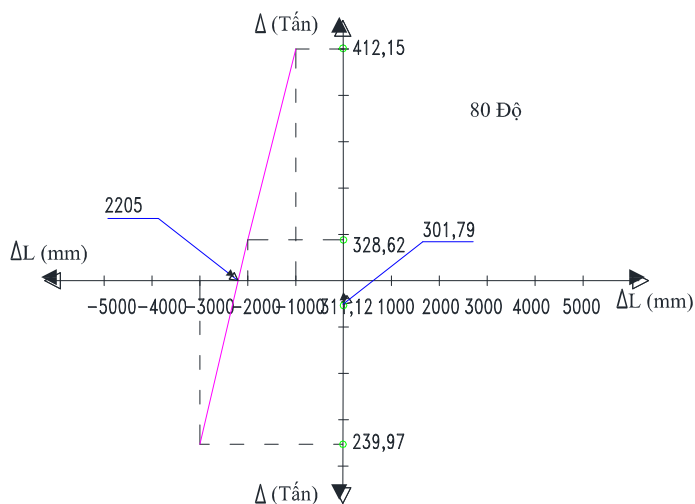


Figure 13 Interpolation graph when $\theta = 80^\circ$

3.9 Trimaran stability graph

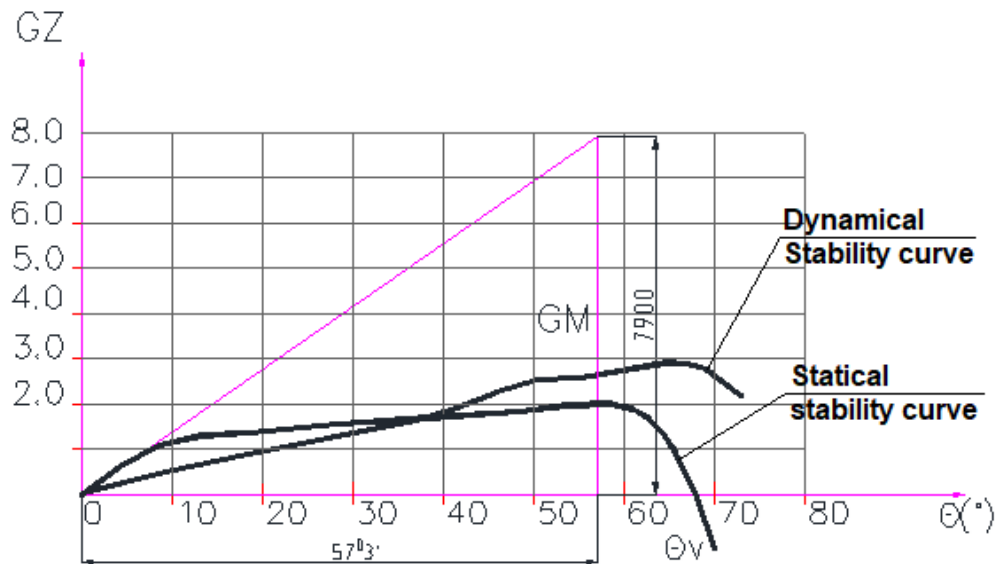


Figure 14 Stability graph of the designed trimaran

IV. RESULTS DISCUSSION

Based on the stability graph of the designed trimaran, the authors determined the value of GM and it turned out that the designed trimaran has a very high initial stability ($GM = 7900\text{mm}$).

The stability graph also shows that when the trimaran starts to tilt from 0° to 10° , its stability lever arm increases very fast, and after 10° , it increases more slowly and holds the high value consistently. After 60° , the lever arm drops very quickly to the state where the ship loses its stability.

High stability leads to a short swing cycle of the ship, which can affect the health of passengers. However, in the case the ship is designed for tourism development in Da Nang, the ship will mostly be used in calm waters, so the high stability of the designed trimaran became the most significant advantage.

V. CONCLUSION

The method of stability calculation used in this research is a combination of traditional method and advanced technology, which helped to improve the calculation accuracy, at the same time, it also saves calculation time very much.

Based on the results of the calculation, the authors concluded that the designed trimaran for Da Nang region of Vietnam has very high stability. It accurately reflects the advantages of a multihull ship compared with a mono-hull one.

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